

Thermal Analysis of Cryogenic Turboexpander with Blade Angle 35° by Using CFD

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Abstract – An attempt of study is made to understand the change in thermal properties occurs in same process when the fluids are changed. Here in our present research work we are analysing the variation occurs in a turbine model with blade angle 35° which is created in Creo element/pro-e by using available data of blade profile later which is simulated in Ansys fluent 14.5 by meshing the model, to specify the simulation settings and physical parameters required to describe the flow through turbo expander at inlet and to examining and analyzing the output results. The variation of minimum temperature, pressure, velocity and entropy are indicated by plotting various graphs with usage of two different fluids i.e., oxygen and nitrogen and by using this result we can analyze flow through expansion turbine.

Index Terms – Turbine, Blade angle 35° , Pro-e, Ansys Fluent 14.5.

1. INTRODUCTION

In our current research, we are majorly focusing upon variation in properties of cryogenics liquids which are widely used in all areas manufacturing, medical, transportation, gas production, oil production and for refrigeration also. For production of this cryogenics liquids it will be done in high pressure plants but for the production low pressure liquids, cryogenic turboexpander is used which consist of compressor, nozzle, expander and exhaust. Here the cryogenic expander is a heart of turbo expander in this phase change will be taken care.

A rotating turbine is used as a turbo expander which is a pressure let-down device that produces low temperatures for the formation of cryogenics liquids this are majorly used in all areas of the gas and oil industries to produce cryogenic refrigeration. Liquefaction of gases is generally used in large scale transportation, storage and low temperature applications. For the production of cryogenic liquids like nitrogen, oxygen and argon in a large scale, cryogenic turbo expander provides the most prominent economical route. Cryogenic turbo expander has the high thermal efficiency and thermal reliability when compared to the high and medium pressure systems.

2. LITERATURE REVIEW

The major component of any cryogenic plant is turbine or the turbo expander. As the major role of turbo expander is to phase change from gas to cryogenics liquids by decreasing static temperature below the refrigeration point. This concept was first introduced by Lord Rayleigh in June 1898. He discussed regarding use of turbine as a expansion device in place of other expansion device. In 1898, a British engineer named Edgar C. Thrupp patented a liquefying machine using an expansion turbine. A simple method sufficient for the design of a high

efficiency expansion turbine is outlined by Kun et. al [1-2].

Agahiet. al. [3-4] have explained the design process of the turboexpander utilizing modern technology, such as CFD software, Computer Numerical Control Technology and Holographic Techniques to further improve an already impressive turboexpander efficiency performance. Several characteristics values are used for defining significant performance criteria of turbo machines such as turbine velocity ratio, pressure ratio, flow coefficient factor and specific speed [5]. Balje has presented a simplified method for computing the efficiency of radial turbo machines and for calculating their characteristics [6]. The concept of specific speed was first introduced for classifying hydraulic machines. Balje [7] introduced this parameter in design of gas turbines and compressors. PhD dissertation of Ghosh [8] explains the detailed summary of technical features and experimental analysis of cryogenic turboexpander. S.K. Ghosh, R.K. Sahoo, S.K. Sarangi in 2005 gave a computational approach to the design of a cryogenic turbine blade profile [9].

3. COMPUTATIONAL FLUID FLOW ANALYSIS

The computational analysis of turbine is majorly done in following three steps. Creo element/pro-E is used to create the model of turbine using available data of hub, shroud and blade profile. ANSYS FLUENT 14.5 is used to mesh the model, to specify the simulation settings and physical parameters required to describe the flow through turbo expander at inlet and to examining and analyzing the output results. The present design procedure is available in literature [8].

4. DESIGNING OF A TURBINE MODEL

Designing of model is done in Pro-E by using data available for blade profile, hub, and shroud. The hub and the tip streamlines are taken from the literature [8,

9]. The surface of turbine model which had generated is considered as the mean surface within a blade. The blade faces will be merged where they are tangent to one another. Here while developing the model there is a need to create a stage fluid zone body for the flow passage, create fluid zone property is selected and an enclosure feature to subtract the blade body. This resulting enclosure can be used for a CFD analysis of the blade passage. Create all blades this property is used to create. All the blades are created using create all blades property by specifying the number of blades in the Pro-E model. Here we are using ten numbers of blades.

In summary, the major dimensions for prototype turbine have been computed by S.K. Ghosh [8] as follows:

Rotational speed: $N = 22910 \text{ rad/sec} = 218790 \text{ rpm}$

Wheel diameter: $D_2 = 16.0 \text{ mm}$

Eye tip diameter: $D_{tip} = 10.8 \text{ mm}$

Eye hub diameter: $D_{hub} = 4.6 \text{ mm}$

Numbers of blades: $Z_{tr} = 10$

Thickness of blades: $t_{tr} = 0.6 \text{ mm}$

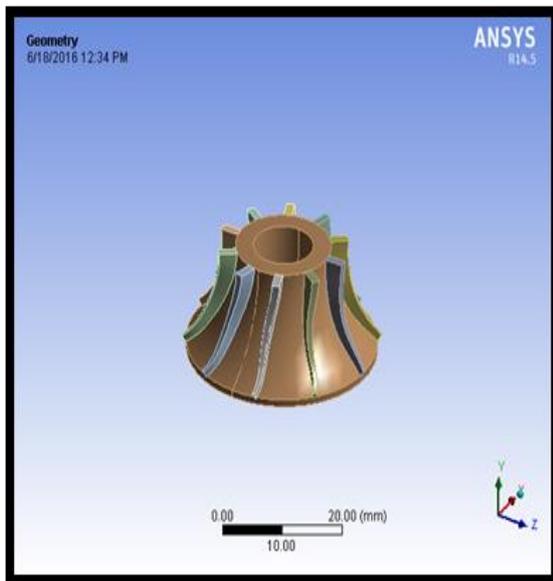


Figure 1: Sample Model designed with using blade angle 35°

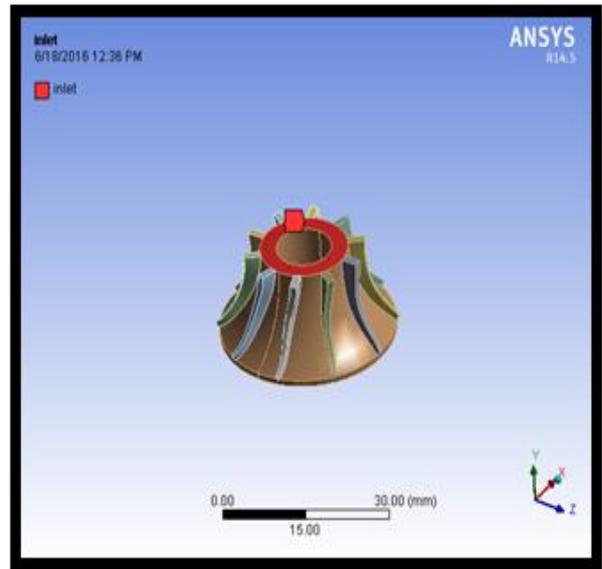


Figure 2: Inlet fluid passage

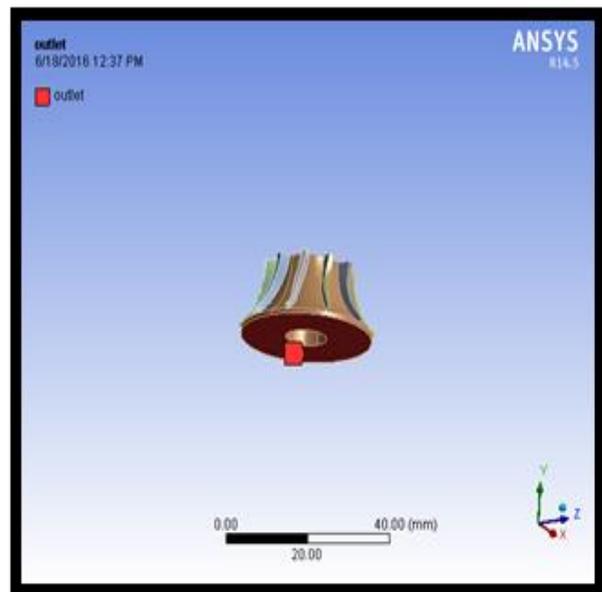


Figure 3: Outlet fluid passage

5. MATERIAL SELECTION

As per the requirement the following material are used in concerned percentage for manufacturing of turbo expander which can sustain in the required condition.

Table 1: Percentage of materials used in turbo expander

Alloying Element	Percentage (%)
Copper	0.1
Iron	0.6
Magnesium	0.4 – 1.4
Manganese	0.4 – 1.0
Chromium	0.3
Silicon	0.6 – 1.3
Aluminum	Rest

6. MESHING AND ITERATION

Ansys fluent 14.5 is used for meshing of turbine model. High quality hexahedral meshes are created in rotating turbine rotor for fluid flow analysis. Geometry information regarding rotating turbine rotor is imported from Pro-e in IGES format. Ansys uses this pro-e file to the length unit, number of blades, axis of rotation and other parameters.

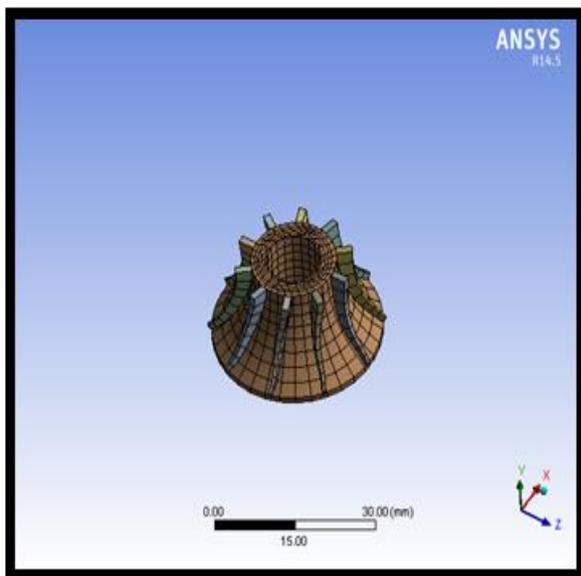


Figure 4: Meshed turbine model.

7. INPUT CONDITIONS FOR ROTATING TURBINE SIMULATION

CFD simulation for rotating turbine is done in Ansys fluent 14.5. In this the model is taken from Pro-e which is converted from 2D to 3D and later it is meshed by fixing the nodes, in software general conditions are provided for pressure based and energy equation with fluid density and viscosity. In following figure, we can see the boundary conditions and fluid condition while inlet of the rotating turbine.

Mass flow rate=0.024 kg/sec

Temperature: 120 K

Pressure= 30 bar

Here we are giving the no. of iteration is 20.

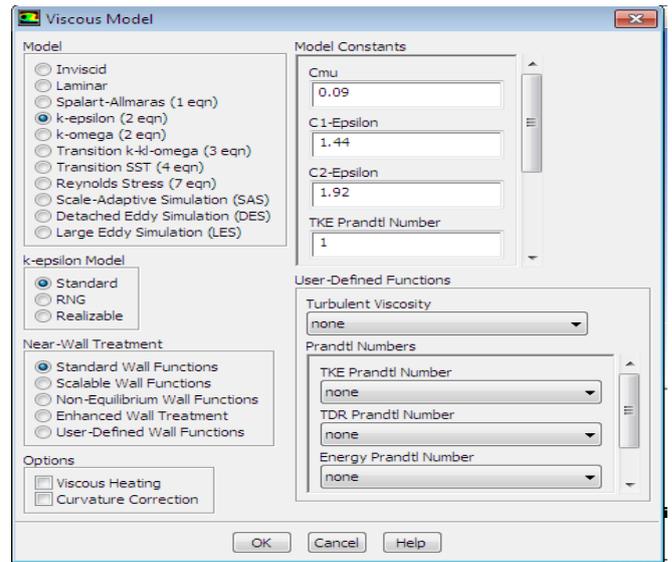


Figure 5: Inlet condition for turbine in Ansys

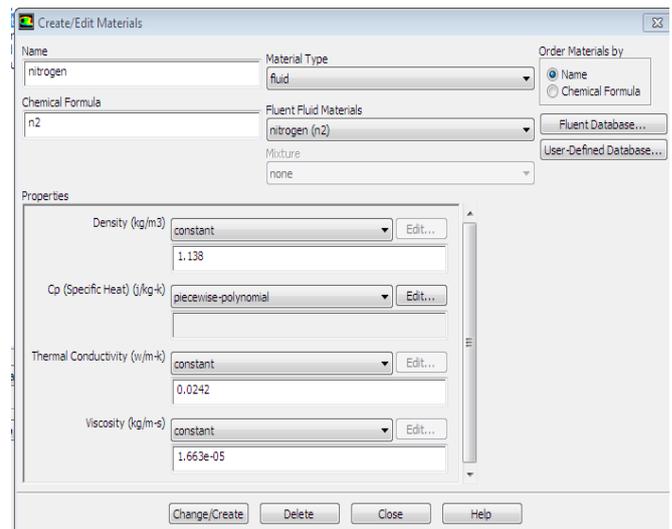


Figure 6: Fluid conditions for nitrogen

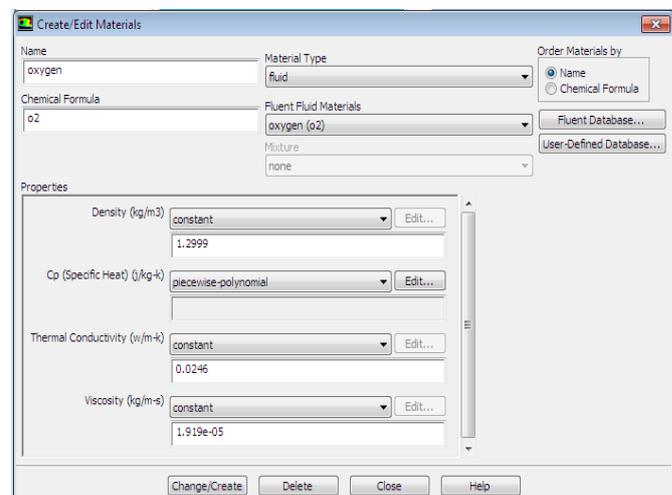


Figure 7: Fluid conditions for oxygen

8. OBTAINED RESULTS

Ansys fluent 14.5 is used to allow easy visualization and quantitative analysis of results of CFD simulations. It can create user defined scalar and vector variables it also includes automatic report.

WITH BLADE ANGLE = 35°
WORKING FLUID = NITROGEN

8.1.1 Pressure

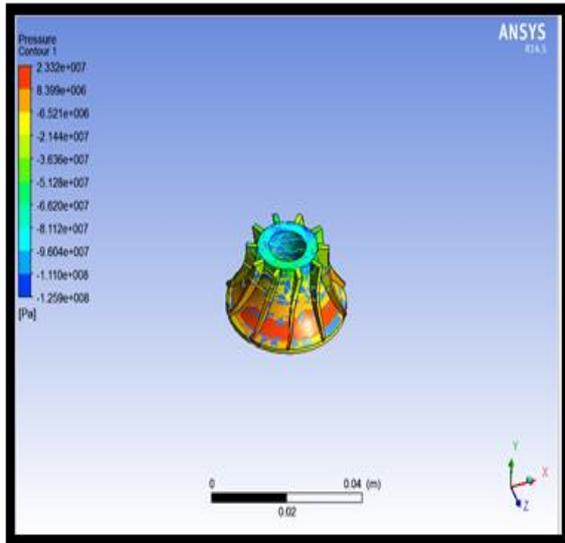


Figure 8: Result of pressure flow inside the turbine

8.1.2 Velocity

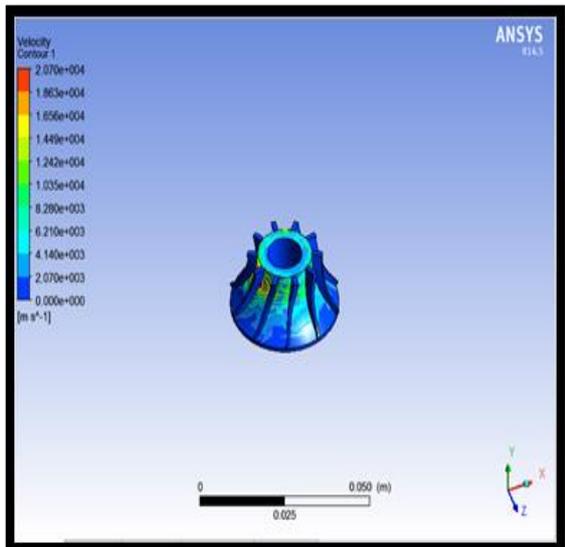


Figure 9: Results of velocity flow inside the turbine

8.1.3 Temperature

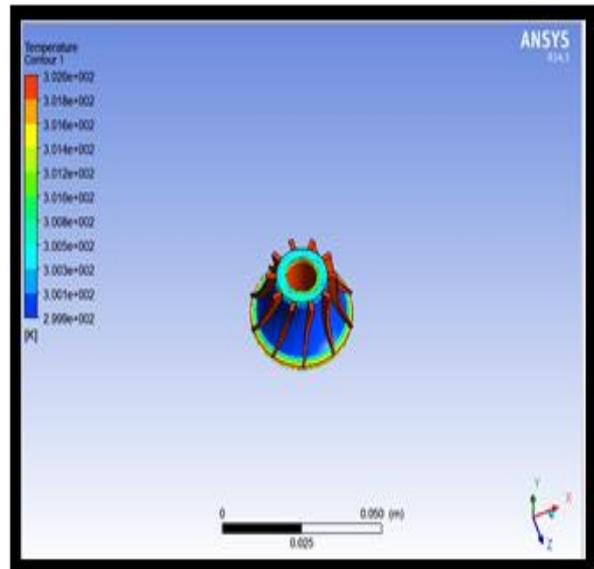


Figure 10: Results of Temperature flow inside the turbine

8.1.4 Entropy

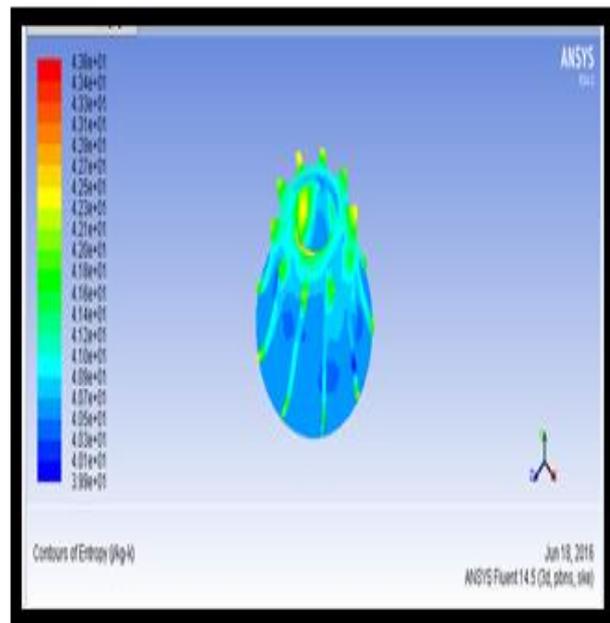


Figure 11: Results of Entropy flow inside the turbine

WITH BLADE ANGLE = 35°
WORKING FLUID = OXYGEN

8.2.1 Pressure

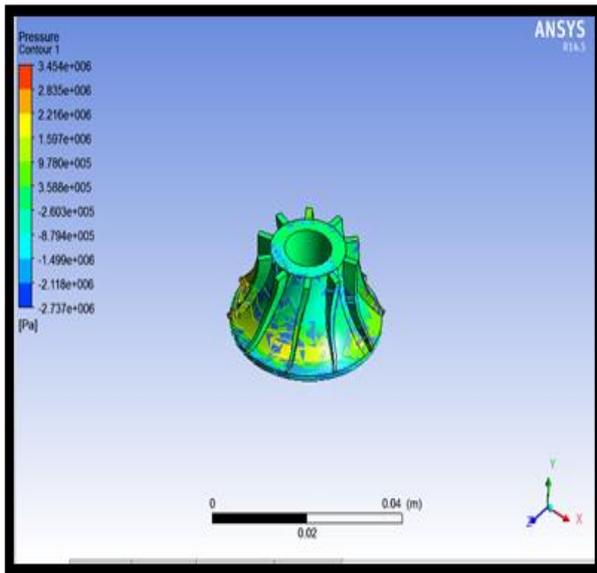


Figure 12: Results of pressure flow inside the turbine

8.2.3 Temperature

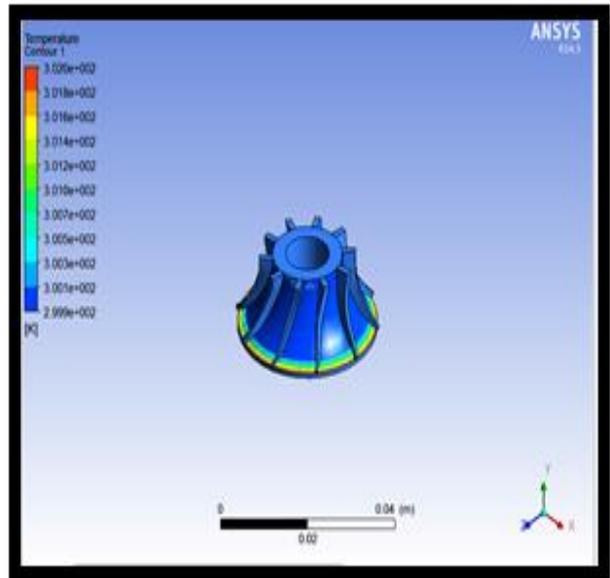


Figure 14: Results of Temperature flow inside the turbine

8.2.2 Velocity

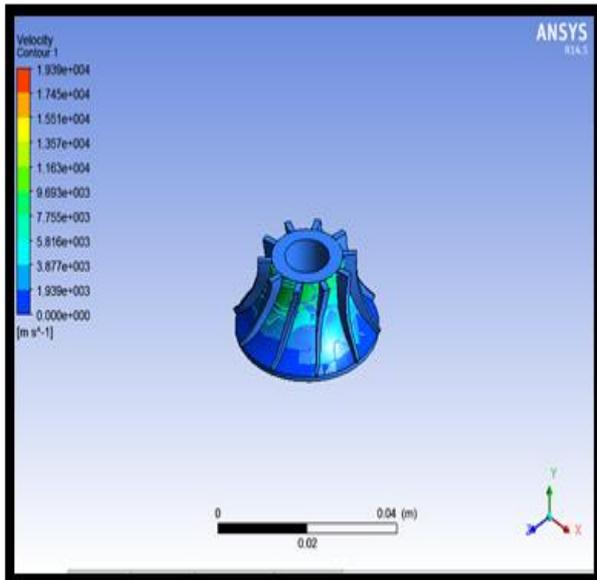


Figure 13: Results of velocity flow inside the turbine

8.2.4 Entropy

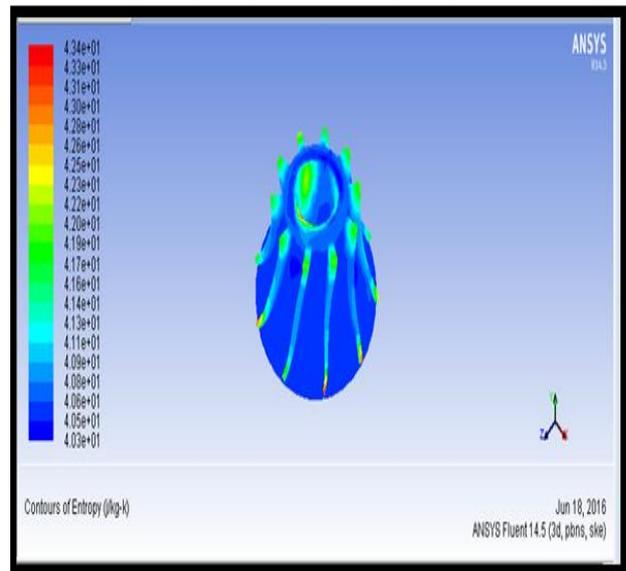


Figure 15: Results of Entropy flow inside the turbine

9. RESULTS & DISCUSSION

Graph and table are available from generated results. In this the input parameters for nitrogen and oxygen fluid provided were same. the input pressure was 30bar but if we see the variation in output pressures, for oxygen fluid output pressure is directly double from the nitrogen fluid. Same in the case of output velocities in some extent we can say.

Table 2: Table for output results

Properties at blade angle 35°	Pressure (Pa)	Velocity (m/s)	Temperature (K)	Entropy (kJ/kg K)
Nitrogen	113	25.44	22	11.85
Oxygen	63	13.93	22	12.6

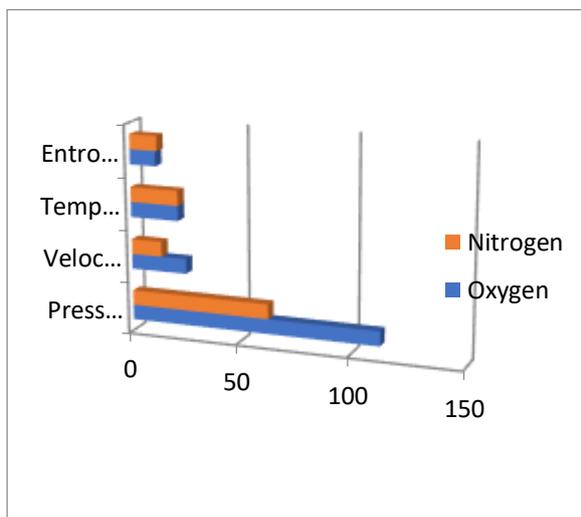


Figure16: Graph shows variation in output results for nitrogen and oxygen

10. CONCLUSION

In this work, the flows analysis is done for rotating turbine of blade angle 35° to understand the variation in performance of turbine during the phase change process and output results when we are using two different fluids i.e., nitrogen and oxygen. Here this rotating turbine model is meshed and simulated for specified inlet conditions which describe the flow velocity, pressure and temperature. On the basis of these inlet parameters that we are analyzing, output results are obtained for which graphs will be drawn and variation is analyzed.

There is a lot of variation in results when fluid is change, for this again the redesigning of turbine model is necessary by changing its blade angle in future work the remodeling and analyzing of rotating turbine can be done by using different blade angle.

Acknowledgments

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